



⑫

## EUROPEAN PATENT SPECIFICATION

⑯ Date of publication of patent specification :  
07.07.93 Bulletin 93/27

㉑ Application number : 89905922.4

㉒ Date of filing : 23.03.89

㉓ International application number :  
PCT/US89/01186

㉔ International publication number :  
WO 89/10495 02.11.89 Gazette 89/26

㉕ Int. Cl.<sup>5</sup> : F16D 65/16, F16D 63/00,  
B60T 13/74, F16D 55/224,  
H02K 7/06, F16H 25/20,  
F16D 59/00

### ㉖ ELECTRICALLY OPERATED DISC BRAKES.

㉗ Priority : 19.04.88 US 183129

㉘ Date of publication of application :  
19.12.90 Bulletin 90/51

㉙ Publication of the grant of the patent :  
07.07.93 Bulletin 93/27

㉚ Designated Contracting States :  
DE FR GB

㉛ References cited :  
EP-A- 017 314  
EP-A- 0 141 609  
WO-A-88/04741  
WO-A-88/04741  
DE-A- 3 412 026  
GB-A- 612 462  
GB-A- 1 297 495

㉜ References cited :  
GB-A- 1 309 645  
GB-A- 1 448 101  
GB-B-21 560 21  
US-A- 1 786 996  
US-A- 2 240 043  
US-A- 4 256 206  
US-A-36 235 81

㉝ Proprietor : ALLIED-SIGNAL INC.  
Columbia Road and Park Avenue P.O. Box  
2245R  
Morristown New Jersey 07962-2245 (US)

㉞ Inventor : TAIG, Alistair, Gordon  
24267 High Street  
Edwardsburg, MI 49112 (US)

㉟ Representative : Lejet, Christian et al  
Bendix Europe Service Techniques Service  
Brevets 126 rue de Stalingrad  
F-93700 Drancy (FR)

EP 0 402 421 B1

Note : Within nine months from the publication of the mention of the grant of the European patent, any person may give notice to the European Patent Office of opposition to the European patent granted. Notice of opposition shall be filed in a written reasoned statement. It shall not be deemed to have been filed until the opposition fee has been paid (Art. 99(1) European patent convention).

**Description**

This invention relates generally to disc brakes that may be operated electrically, and in particular to an electrically operated disc brake containing a high efficiency screw and nut mechanism and/or a one-way clutch mechanism having a unique release feature.

Disc brakes have been utilized for many years in passenger cars, heavy trucks, and aircraft. Because of the increasing emphasis on reducing the weight of vehicles and simplifying the components thereof, it is desirable to develop a braking system that is operated electrically. Such a braking system must be highly reliable, cost effective, and practical within the packaging constraints of the particular vehicle. A prior anti-friction device is shown in US-A-3,614,900, prior clutch mechanisms are shown in GB-A-1,309,645 and US-A-3,623,581, and an improved screw and nut assembly is shown in GB-A-2 156 021. The present invention provides a disc brake that may be operated by an electric motor and which may contain a highly efficient nut and screw mechanism and/or a one-way clutch mechanism having a unique release feature. The high efficiency screw and nut mechanism provides a simplified structure that is highly reliable, low cost, and which will fit readily within the packaging constraints of several vehicles that may utilize an electrically operated disc brake. Likewise, the one-way clutch mechanism with unique release feature provides a highly reliable, low cost clutch mechanism which enables the disc brake to meet Federal requirements for vehicle parking brakes, while at the same time being small enough so that the overall size of the electrically operated disc brake will continue to fit readily within the packaging constraints of several vehicles.

The present invention comprises a disc brake that may be operated by motor means, comprising a caliper having a bore and a piston housing actuatable to displace a pair of friction elements into engagement with a rotor, motor means for actuating operatively a high efficiency screw and nut mechanism, the screw and nut mechanism effective to displace axially said piston housing and cause the displacement of the pair of friction elements, the screw comprising a screw thread having a predetermined pitch and a predetermined pitch diameter, a first nut disposed about said screw and having an interior opening with a nut thread disposed therein, the nut thread engaging at a circumferential portion thereof the screw thread, the nut thread having a pitch equal to the predetermined pitch of the screw thread, and the nut thread having a predetermined pitch diameter greater than the predetermined pitch diameter of the screw thread, a bearing race member disposed within said piston housing and including a bearing race, the first nut including a complementary-shaped bearing race

and bearing means disposed between the bearing races, a second nut disposed within an interior opening of the piston housing, the second nut having an interior opening with a second nut thread disposed thereabout, the second nut thread engaging at a circumferential portion thereof the thread of the screw, the second nut thread having a pitch equal to the predetermined pitch of the screw thread and a pitch diameter greater than the predetermined pitch diameter of the screw thread, a thrust race member disposed within the interior opening of the piston housing, the thrust race member disposed axially between the nuts and including therein a bearing race, the second nut having a complementary-shaped bearing race and a bearing mechanism disposed between the bearing races of the thrust race member and second nut, and a sleeve member disposed between the thrust race member and bearing race member, so that rotation of the screw effects rotation and axial displacement of the second nut against the thrust race member which transmits axial displacement, via the sleeve member, to the bearing race member and piston housing. The present invention also comprises a disc brake that may be operated by motor means, comprising a caliper having a bore with an actuating mechanism disposed therein, the actuating mechanism and caliper actuatable to displace a pair of friction elements into engagement with a rotor, the actuating mechanism including a one-way clutch mechanism connected with drive shaft of said motor means, the clutch mechanism comprising the drive shaft having a transverse drive pin extending therefrom, the drive shaft and the drive pin received within a drive member which has radially extending drive slots, the drive pin extending through said drive slots, a cage member disposed about said drive member and having therein a pair of oppositely disposed radial cage slots, the drive pin extending into the radial cage slots, the drive member including cam surfaces thereabout receiving bearing means thereat, and a housing disposed about said cage member and having an interior surface engaged by said bearing means, whereby rotation in one direction of said drive member by the actuating mechanism causes said bearing means to engage between said cam surfaces and interior surface of the housing in order to fix non-rotatably the drive member, and rotation of the drive member by the drive shaft in the one direction causes said drive pin to engage radial abutment surfaces of the radial cage slots to effect rotation of the cage member and cause said bearing means to move to non-engagement with said housing and permit continual rotation of said drive member in the one direction to effect operation of the actuating mechanism.

The invention is described in detail below with reference to the drawings which illustrate embodiments in which:

Figure 1 is a section view of an electrically oper-

ated disc brake having a high efficiency screw and nut mechanism;

Figure 2 is a section view of an electrically operated disc brake having a one-way clutch mechanism with unique release feature; and

Figure 3 is a section view taken along view line 3-3 of Figure 2.

A disc brake of the present invention is referenced generally by numeral 10 in Figure 1. Disc brake 10 comprises a brake that is operated by motor means 40. Motor means 40 may comprise an electric motor or other motor mechanisms that provide a rotary output via shaft 41. Disc brake 10 includes a caliper 12 having a caliper housing 14 with a bore 16. Caliper 12 extends over a pair of friction elements 18 and 20 which may be displaced toward one another in order to brake a rotor 22. The bore 16 comprises a groove 17 adjacent opening 19 and bore end 21. Seal 32 is disposed within groove 17. Located within bore 16 is an actuating mechanism referenced generally by numeral 90. Actuating mechanism 90 comprises a planetary gear assembly 50 which includes a sun gear 52, three planetary gears 54 (the other two not shown in the section view), a carrier 55, pins 53 which carry the planetary gears, two ring gears 70, 80, and thrust bearings 75. Each ring gear has internal teeth, the ring gear 80 being rotatable but having fewer teeth than ring gear 70 which is operatively coupled with caliper housing 14. The planetary gear assembly 50 comprises the planetary gear assembly disclosed in copending patent Application Serial No. 946,400

entitled "Electrically Operated Disc Brake" which is incorporated by reference herein. Because the two ring gears 70 and 80 have different numbers of teeth, the planetary gear assembly has a high reduction ratio, as disclosed in copending Serial No. 946,400. Rotatable ring gear 80 has a recess 82 with splines 84 therein. A screw 86 includes a first end 87 having splines 88 which engage the splines 84 of rotatable ring member 80 and thereby couple together nonrotatably the screw 86 and ring member 80. First end 87 and recess 82 are curved so that the screw may self-center itself relative to ring gear 80. Located within bore 16 is a piston housing 68 which has an interior opening 69. piston housing 68 includes a key 71 received within slot 31 of friction element 18 so that the two are coupled nonrotatably together. The high efficiency screw and nut mechanism is referenced generally by numeral 60. Disposed about screw 86 is at least one nut 77. Nut 77 has an interior opening 78 containing a nut thread 79. Nut thread 79 engages, at a circumferential portion thereof, a screw thread 89. The nut 77 includes a bearing race 73 which receives ball bearings 100. Located at the closed end of piston housing 68 is a bearing race member 91 which includes a bearing race 92 that receives the bearings 100. Second nut 97 includes an interior opening 98 having a second nut thread 99 which engages at a cir-

cumferential portion thereof the screw thread 89. Second nut 97 includes a race 96 which receives bearing members 101 that are seated within the correspondingly shaped race 106 of thrust race member 95. Thrust race member 95 abuts a sleeve 105 that also abuts at the other end the bearing race member 91. Disposed between nut 77 and sleeve 105 is a needle type clutch 107. Clutch 107 permits nut 77 to rotate freely in only one direction of rotation but eliminating, absent rotation of the screw, rotation of the nut in the other direction.

It is desirable to use a highly efficient screw and nut mechanism which, in at least one direction of rotation, minimizes as much as possible the frictional contact between threads 89 of screw 86 and nut threads 79, 99. In order to accomplish this, screw threads 89 and nut threads 79, 99 have the same pitch (see Distance A). However, the pitch diameter of nuts 77, 97 (as indicated by Distance B on nut 97) is slightly greater than the pitch diameter C of screw 86. Thus, the pitch diameters of nuts 77 and 97 may be equal to each other or slightly different, but the pitch diameters of both nuts are at least slightly greater than the pitch diameter of screw 86, which results in the rotation of nuts 77, 97 at a speed different than the rotation of screw 86. Nut 97 engages bearings 101 that engage the thrust race member 95, and thrust race member 95 engages sleeve 105 in order to transmit thrust to the bearing race member 91 and piston housing 68. As nuts 77, 97 advance axially and slowly along screw 86, the nuts engage the respective bearings 100, 101. A balancing of the mechanism is accomplished by bearings 100 and 101 being offset and the threads 79 and 99 engaging opposite radial sides of screw thread 89.

The electric motor 40, in response to signals from a controller (not shown), drives sun gear 52 of planetary gear mechanism 50. As planetary gear mechanism 50 effects the rotation of rotatable ring gear 80, rotatable ring gear 80 rotates screw 86 which, through the difference in pitch diameters B and C, causes nuts 77, 97 to rotate and advance slowly along screw thread 89. The substantial advantage of the present invention is that circumferential sliding friction between the screw and nuts is virtually eliminated, and only low velocity radial sliding will occur as the screw and nuts rotate together. A very high mechanical efficiency is achieved so that a fully reversible action is possible, and is thus suitable for a service brake actuator. It is desirable to provide a mechanism that may be utilized as a parking brake mechanism wherein the same high efficiency can be achieved to apply the brake, but a low efficiency in the reverse drive direction be provided in order to allow the actuator to hold in its applied position until reverse motor torque is effected. To accomplish this, the needle type clutch 107 allows nut 77 to rotate freely in one direction, but prevents and eliminates any free rotation of the nut in the

reverse direction of rotation. Thus, in order to release disc brake 10, screw 86 must be rotated in the reverse direction in order to cause sliding between screw 86 and nut 77, 97. By suitable selection of the pitch of the threads of the screw and nuts, the brake mechanism can be made irreversible, all without affecting the application efficiency of the brake. Because the differential screw and nut mechanism is utilized in order to advance the nut or nuts slowly along the screw thread, a lower ratio reduction gear box may be utilized for the driving of the screw. The screw 86 functions like a fine pitch screw but without the high frictional losses associated with fine pitch screw threads.

Figures 2 and 3 illustrate an embodiment in which the disc brake is referenced generally by numeral 210. Similar structure will be indicated by the same reference numeral increased by 200. The electric motor 240 drives a planetary gear assembly 250 which is housed within the caliper housing 214 of caliper 212. The planetary gear assembly 250 includes a sun gear drive member 252 which drives a carrier 255, pins 263, and planetary gears 254. The fixed ring gear 270 nonrotatably engages the caliper housing 214 by means of the housing shoulder 213 received within the gear recess 271. Rotatable ring gear 280 includes a helical groove 289 with bearing balls 285 disposed therein and engaging helical grooves 269 of the actuator sleeve 268. The structure of the planetary gear assembly 250, ring gears 270 and 280, and actuator sleeve 268 is similar to that disclosed in co-pending Patent Application Serial no. 105,756 entitled "Electrically Actuated Disc Brake" which is incorporated by reference herein. The ring gears have a tooth differential, with rotatable ring gear 280 having fewer teeth than ring gear 270, so that planetary gear assembly 250 provides a high reduction ratio.

The one-way clutch mechanism with unique release feature is designated generally by reference numeral 230. Electric motor 240 includes a motor drive shaft 241 that includes transverse opening 242 (see Figure 2). Transverse opening 242 receives therein a drive pin 243. Drive pin 243 is also received within the sun gear drive member 252 (see Figure 3). Sun gear drive member 252 has a bore 251 which receives the motor drive shaft 241. Sun gear drive member 252 includes oppositely disposed radial drive slots 257 which have abutment edges 257a and 257b. Drive member 252 is journaled within a cage member 281. Cage member 281 includes oppositely disposed radial cage slots 282 which have abutment surfaces 283. Cage member 281 also includes oppositely disposed radial openings 284. About the periphery of sun gear drive member 252 are cam surfaces 253 which receive roller bearings 290. Roller bearings 290 extend through radial openings 284 of cage member 281. Drive pin 243 extends through the radial drive slots 257 and into the radial cage slots 282. Drive slots 257

are in circumferential nonalignment with the radial cage slots 282. Disposed about cage member 281 is an annular housing 295 which extends into radially extending flange portions 296 that are fixed to caliper housing 214. The interior surface 297 of housing 295 is engaged by roller bearings 290. Planetary gear assembly 250 is driven by electric motor 240 via the one-way clutch mechanism 230 which prevents a back drive of the motor by mechanism 290. A reversal of the electric motor direction of rotation automatically unlocks clutch mechanism 230 and allows reverse rotation of drive shaft 241 as long as electric motor 240 is turning drive shaft 241. Clutch mechanism 230 may be used in place of the one-way roller clutch 107 illustrated in the electric disc brake of Figure 1. The utilization of clutch mechanism 230 to prevent back-off of the brake mechanism allows high efficiency screw mechanisms such as those described above to be used with a consequent savings in motor power and the sizes of component parts.

When motor 240 operates and rotates motor shaft 241 in the counterclockwise direction (see Arrow A in Figure 3), drive pin 243 engages abutment surfaces 257a of sun gear drive member 252 (as illustrated) and causes roller bearings 290, as member 252 rotates relative to cage member 281, to move to the deep ends of cam surfaces 253 where there is sufficient radial clearance to prevent jamming of roller bearings 290 between the interior surface 297 of housing 295 and cam surfaces 253. Thus, motor 240 can drive sun gear drive member 252 in a counterclockwise direction and effect operation of planetary gear assembly 250 and axial displacement of friction element 218 into engagement with rotor 222 and, by reaction, the axial displacement of friction element 220 into engagement with the other face of rotor 222. When electric current to motor 240 is terminated, reaction forces cause the actuator sleeve 268 and rotatable ring member 280 to turn sun gear drive member 252 in a clockwise direction. Drive member 252 can rotate slightly in the clockwise direction (see Arrow B) until roller bearings 290 ride up cam surfaces 253 and move radially outwardly so that they become wedged between cam surfaces 253 and interior surface 297 of housing 295. Thus, sun gear drive member 252 cannot back drive motor 240 and brake 210 remains mechanically locked in the applied position. If electric motor 240 receives a signal which requires rotation of drive member 252 in the clockwise direction (Arrow B), motor 240 initially rotates pin 243 relative to drive member 252, drive shaft 241 being journaled within bore 251 of drive member 252. Drive pin 243 rotates in the clockwise direction within radial cage slots 282 and radial slots 257 such that the drive pin engages abutment surfaces 283 of cage member 281 before the pin ever engages abutment surfaces 257b of drive member 252. This causes cage member 281 to rotate relative to member 252, the rotation of

cage member 281 causing roller bearings 290 to rotate in the clockwise direction and move into the radially recessed portions of cam surfaces 253. This effects an unlocking of drive member 252 relative to housing 295, and drive member 252 can be driven in the clockwise direction as pin 243 then engages surface 257b, and continuously driven as long as electric motor 240 is rotating drive shaft 241. When motor 240 terminates its rotation of drive shaft 241, roller bearings 290 can immediately reengage the radially outer portions of cam surfaces 253 and become wedged between the radially outer portions of the cam surfaces and interior surface 297 of fixed housing 295. This locks sun gear drive member 252 relative to fixed housing 295. Thus, electric motor 240 is utilized to apply and release brake 210, and with only minimal electric power utilized for release of the brake. The one-way clutch mechanism 230 with unique release feature can be utilized equally well for a service or parking brake application and has the advantage of utilizing high motor power only when the braking force is being increased. When steady braking is occurring, the clutch mechanism 230 will remain applied and no further motor power is needed. Motor power need only be applied in order to either apply the brake further or to effect a release of the brakes via a release of the clutch mechanism.

The one-way clutch mechanism with unique release feature enables the brake to be highly efficient for either parking or service brake applications, and with a consequent savings in energy requirements for actuation.

The drive motor is able to control both the application and release of the brake. The one-way clutch mechanism operated by the motor is very compact, and has a low cost.

### Claims

1. A disc brake that may be operated by motor means, comprising a caliper (12) having a bore (16) and a piston housing (68) actuatable to displace a pair of friction elements (18, 20) into engagement with a rotor (22), motor means (40) for actuating operatively a high efficiency screw and nut mechanism (60), the screw and nut mechanism (60) effective to displace axially said piston housing (68) and cause the displacement of the pair of friction elements (18, 20), the screw (86) comprising a screw thread (89) having a predetermined pitch and a predetermined pitch diameter, a first nut (77) disposed about said screw and having an interior opening (78) with a nut thread (79) disposed therein, the nut thread (79) engaging at a circumferential portion thereof the screw thread (89), the nut thread (79) having a pitch equal to the predetermined pitch of the screw

5 thread (89), and the nut thread (79) having a predetermined pitch diameter greater than the predetermined pitch diameter of the screw thread (89), a bearing race member (91) disposed within said piston housing (68) and including a bearing race (92), the first nut (77) including a complementary-shaped bearing race (73) and bearing means (100) disposed between the bearing races (73, 92), a second nut (97) disposed within an interior opening (69) of the piston housing (68), the second nut (97) having an interior opening (98) with a second nut thread (99) disposed thereabout, the second nut thread (99) engaging at a circumferential portion thereof the thread (89) of the screw (86), the second nut thread (99) having a pitch equal to the predetermined pitch of the screw thread (89) and a pitch diameter greater than the predetermined pitch diameter of the screw thread (89), a thrust race member (95) disposed within the interior opening (69) of the piston housing (68), the thrust race member (95) disposed axially between the nuts (77, 97) and including therein a bearing race (106), the second nut (97) having a complementary-shaped bearing race (96) and a bearing mechanism (101) disposed between the bearing races (96, 106) of the thrust race member (95) and second nut (97), and a sleeve member (105) disposed between the thrust race member (95) and bearing race member (91), so that rotation of the screw (86) effects rotation and axial displacement of the second nut (97) against the thrust race member (95) which transmits axial displacement, via the sleeve member (105), to the bearing race member (91) and piston housing (68).

10

15

20

25

30

35

40

45

50

55

2. The disc brake in accordance with claim 1, wherein the disc brake comprises a clutch (107) disposed between the first nut (77) and sleeve member (105), the clutch (107) permitting said first nut (77) to rotate freely in one direction and preventing rotation, absent rotation of the screw (86), in an opposite direction.

3. The disc brake in accordance with Claim 1, further comprising a one-way clutch mechanism (230) connected with a drive shaft (241) of said motor means (240), the clutch mechanism (230) comprising the drive shaft (241) having a transverse drive pin (243) extending therefrom, the drive shaft (241) and the drive pin (243) received within a drive member (252) which has radially extending drive slots (257), the drive pin (243) extending through said drive slots (257), a cage member (281) disposed about said drive member (252) and having therein a pair of oppositely disposed radial cage slots (282), the drive pin (243) extending into the radial cage slots (282), the

drive member (252) including cam surfaces (253) thereabout receiving bearing means (290) thereat, and a housing (295) disposed about said cage member (281) and having an interior surface (297) engaged by said bearing means (290), whereby rotation in one direction (B) of said drive member (252) by the actuating mechanism (90, 230) causes said bearing means (290) to engage between said cam surfaces (253) and interior surface (297) of the housing (295) in order to fix nonrotatably the drive member (252), and rotation of the drive member (252) by the drive shaft (241) in the one direction (B) causes said drive pin (243) to engage radial abutment surfaces (283) of the radial cage slots (282) to effect rotation of the cage member (281) and cause said bearing means (290) to move to non-engagement with said housing (295) and permit continual rotation of said drive member (252) in the one direction (B) to effect operation of the actuating mechanism (90, 230).

4. The disc brake in accordance with claim 3, wherein the cage member (281) has a pair of oppositely disposed openings (284) in which is disposed said bearing means (290) which engages said cam surfaces (253) of the drive member (252).

5. The disc brake in accordance with claim 4, wherein the radial cage slots (282) are in circumferential nonalignment with said drive slots (257).

6. The disc brake in accordance with claim 5, wherein the cam surfaces (253) are disposed radially inwardly of an outer peripheral surface of said drive member (252).

7. The disc brake in accordance with claim 6, wherein the bearing means (290) comprises a plurality of roller bearings (290).

8. The disc brake in accordance with claim 7, wherein the drive member (252) includes an interior bore receiving rotatably therein said drive shaft (241).

5. Kolbengehäus (68) axial verschieben und die Verschiebung des Paares von Reibelementen (18, 20) bewirken kann, wobei die Schraube (86) ein Schraubengewinde (89) mit einer vorgegebenen Ganghöhe und einem vorgegebenen Flankendurchmesser besitzt, einer ersten Mutter (77), die um die Schraube herum angeordnet ist und eine Innenöffnung (78) mit einem darin befindlichen Muttergewinde (79) aufweist, das an einem Umfangsabschnitt mit dem Schraubengewinde (89) kämmt und eine Ganghöhe aufweist, die der vorgegebenen Ganghöhe des Schraubengewindes (89) entspricht, sowie einen vorgegebenen Flankendurchmesser aufweist, der größer ist als der vorgegebene Flankendurchmesser des Schraubengewindes (89), einem Lagerringelement (91), das innerhalb des Kolbengehäuses (68) angeordnet ist und einen Lagerring (92) besitzt, wobei die erste Mutter (77) einen komplementär ausgebildeten Lagerring (73) und zwischen den Lagerringen (73, 92) angeordnete Lagereinrichtungen aufweist, einer zweiten Mutter (97), die innerhalb einer Innenöffnung (69) des Kolbengehäuses (68) angeordnet ist und eine Innenöffnung (98) mit einem durch diese herum angeordneten zweiten Muttergewinde (99) aufweist, wobei das zweite Muttergewinde (99) an einem Umfangsabschnitt mit dem Gewinde (89) der Schraube (86) kämmt und eine Ganghöhe aufweist, die der vorgegebenen Ganghöhe des Schraubengewindes (89) entspricht, sowie einen Flankendurchmesser, der größer ist als der vorgegebene Flankendurchmesser des Schraubengewindes (89), einem Axialdruckringelement (95), das innerhalb der Innenöffnung (69) des Kolbengehäuses (68) und axial zwischen den Muttern (77, 97) angeordnet ist und einen Lagerring (106) aufweist, wobei die zweite Mutter (97) einen komplementär ausgebildeten Lagerring (96) und einen zwischen den Lagerringen (96, 106) des Axialdruckringelements (95) und der zweiten Mutter (97) angeordneten Lagermechanismus (101) besitzt, und einem Hülsenelement (105), das zwischen dem Axialdruckringelement (95) und dem Lagerringelement (91) angeordnet ist, so daß durch eine Drehung der Schraube (86) eine Drehung und Axialverschiebung der zweiten Mutter (97) gegen das Axialdruckringelement (95) bewirkt wird, das die Axialverschiebung über das Hülsenelement (105) auf das Lagerringelement (91) und das Kolbengehäuse (61) überträgt.

6. Scheibenbremse nach Anspruch 1 mit einer Kupplung (107), die zwischen der ersten Mutter (77) und dem Hülsenelement (105) angeordnet ist und ein freies Drehen der ersten Mutter (77) in einer Richtung ermöglicht und bei fehlender Dre-

#### Patentansprüche

1. Scheibenbremse, die durch Motoreinrichtungen betätigt werden kann, mit einem Sattel (12) mit einer Bohrung (16) und einem Kolbengehäuse (68), das zur Verschiebung eines Paares von Reibelementen (18, 20) in Eingriff mit einem Rotor (22) betätigbar ist, Motoreinrichtungen (40) zur Betätigung eines Schrauben/Mutter-Mechanismus (60) mit hohem Wirkungsgrad, der das
2. Scheibenbremse nach Anspruch 1 mit einer Kupplung (107), die zwischen der ersten Mutter (77) und dem Hülsenelement (105) angeordnet ist und ein freies Drehen der ersten Mutter (77) in einer Richtung ermöglicht und bei fehlender Dre-

hung der Schraub (86) ein Drehen der ersten Mutter in der entgegengesetzten Richtung verhindert.

3. Scheibenbremse nach Anspruch 1, die des weiteren einen Einwegkupplungsmechanismus (230) umfaßt, der mit einer Antriebswelle (241) der Motoreinrichtungen (240) in Verbindung steht, wobei die Antriebswelle (241) einen quer verlaufenden Antriebsstift (243) aufweist, der sich hiervon erstreckt, die Antriebswelle (241) und der Antriebsstift (243) innerhalb eines Antriebselementes (242) angeordnet sind, das radial verlaufende Antriebsschlitzte (257) aufweist, sich der Antriebsstift (243) durch die Antriebs schlitzte (257) erstreckt, ein Käfigelement (281) um das Antriebselement (252) herum angeordnet ist und ein Paar von gegenüberliegend angeordneten radialen Käfigschlitzten (282) aufweist, sich der Antriebsstift (243) in die radialen Käfig schlitzte (282) erstreckt, das Antriebselement (252) um es herum Nockenflächen (253) besitzt, die daran Lagereinrichtungen (290) aufnehmen, und ein Gehäuse (295) um das Käfigelement (281) herum angeordnet ist und eine Innenfläche (297) aufweist, mit der die Lagereinrichtungen (290) in Eingriff stehen, so daß durch eine Drehung des Antriebselementes (252) durch den Betätigungsmechanismus (90, 230) in einer Richtung (B) die Lagereinrichtungen (290) zwischen die Nockenflächen (253) und die Innenfläche (297) des Gehäuses (295) greifen, um das Antriebselement (252) nicht drehbar zu fixieren, und durch eine Drehung des Antriebselementes (252) durch die Antriebswelle (241) in der einen Richtung (B) der Antriebsstift (243) mit den radialen Anschlagflächen (283) der radialen Käfigschlitzte (282) in Eingriff tritt, um eine Drehung des Käfigelementes (281) zu bewirken und eine Bewegung der Lagereinrichtungen (290) außer Eingriff mit dem Gehäuse (295) zu veranlassen und eine kontinuierliche Drehung des Antriebselementes (252) in der einen Richtung (B) und hierdurch eine Funktion des Betätigungsmechanismus (90, 230) zu ermöglichen.

4. Scheibenbremse nach Anspruch 3, bei der das Käfigelement (281) ein Paar von gegenüberliegend angeordneten Öffnungen (284) aufweist, in denen die Lagereinrichtungen (290), die mit den Nockenflächen (253) des Antriebselementes (252) in Eingriff stehen, angeordnet sind.

5. Scheibenbremse nach Anspruch 4, bei der die radialen Käfigschlitzte (282) in Umfangsrichtung nicht zu den Antriebsschlitzten (257) ausgerichtet sind.

6. Scheibenbremse nach Anspruch 5, bei der die Nockenflächen (253) radial innerhalb ein räumlichen Umgangsfläche des Antriebselementes (252) angeordnet sind.

7. Scheibenbremse nach Anspruch 6, bei der die Lagereinrichtungen (290) eine Vielzahl von Rollenlagern (290) umfassen.

8. Scheibenbremse nach Anspruch 7, bei der das Antriebselement (252) eine Innenbohrung aufweist, die die Antriebswelle (241) drehbar aufnimmt.

#### Revendications

1. Frein à disque actionné par moteur, comprenant un étier (12) percé d'un alésage(16) et un carter(68) de piston commandé pour engager une paire d'éléments de friction (18,20) avec un rotor (22) , un moteur (40) pour actionner un ensemble vis-écrou (60) à haut rendement, l'ensemble vis-écrou (60) agissant pour déplacer axialement le carter (68) de piston et provoquer le déplacement de la paire d'éléments de friction (18,20), la vis (86) comportant un filet (89) ayant un pas préterminé et un diamètre de pas préterminé, un premier écrou (77) disposé autour de la vis et présentant une ouverture intérieure (78) incluant un filetage (79) , le dit filetage (79) engrenant le filet de vis (89) à sa périphérie, le filetage (79) ayant un pas égal au pas préterminé du filet de vis,et le filetage (79) de l'écrou ayant un diamètre de pas supérieur au diamètre du pas préterminé du filet de vis (89), un organe de roulement 91 disposé dans le carter (68) du piston et incluant un chemin de roulement (92) ,le premier écrou (77) incluant un chemin de roulement de forme complémentaire (73) et un palier(100) disposé entre les chemins de roulement (73,92) , un second écrou (97) disposé dans une ouverture intérieure (69)du carter du piston (68), le second écrou (97) ayant une ouverture intérieure (98) incluant un filetage d'écrou (99) , le filetage (99) du second écrou engranant le filet(89) de la vis (86) à sa périphérie, le filetage (99) du second écrou ayant un pas égal au pas préterminé du filet de vis (89) et un diamètre de pas supérieur au diamètre de pas préterminé du filet de vis (89), un élément de butée (95) disposé dans l'ouverture intérieure (69) du carter de piston (68) ,le dit élément de butée (95) étant disposé axialement entre les écrous (77,97) et incluant un chemin de roulement (106) le second écrou (97) ayant un chemin de roulement de forme complémentaire (96) et un palier (101) disposé entre les chemins de roulement (96,106) de l'élément de poussée

(95) et le second écrou (97) et un manchon (105) disposé entre l'élément de poussée (95) et l'organe de roulement (91), de telle sorte que la rotation d la vis (86) assure la rotation et le déplacement axial du second écrou (97) contre l'élément de poussée (95) qui transmet le déplacement axial, à travers le manchon (105) à l'organe de roulement (91) et au carter du piston (68).

2. Frein à disque selon la revendication 1, dans lequel le frein à disque comprend un embrayage (107) disposé entre le premier écrou (77) et le manchon (105) cet engrenage (107) permettant au premier écrou (77) de tourner librement dans un sens et interdisant la rotation de la vis (86) dans le sens opposé.

3. Frein à disque selon la revendication 1, comprenant en outre un mécanisme d'embrayage à roue libre(230) relié à un arbre d'entraînement (241) du moteur (240) , l'embrayage (230) comprenant l'arbre d'entraînement (241) traversé par une goupille (243) ,l'arbre (241) et la goupille (243) étant reçus dans un organe d'entraînement (252) percé de fentes d'entraînement radiales (257),la goupille d'entraînement (243)traversant les fentes (257), une cage (281)disposée autour de l'organe d'entraînement (252) et incluant une paire de fentes radiales (282)disposées en vis à vis, la goupille d'entraînement (243) s'étendant dans les fentes radiales (282) de la cage, l'organe d'entraînement (252) incluant les surfaces de cames (253) autour recevant le palier à roulement (290) et un carter (295) disposé autour de la cage (281) et présentant une surface interne (297) en prise avec le palier à roulement (290), par lequel la rotation dans le sens (B) de l'organe d'entraînement (252) par le mécanisme d'actionnement (90,230) amène le palier (290) à s'engager entre les surfaces des came (253) et la surface interne (297)du carter (295) de manière à fixer non-rotativement l'organe d'entraînement (252), et que la rotation de cet organe par l'arbre d'entraînement (241) dans le sens (B) amène la goupille (243) en prise avec les surfaces de butée radiales(283) des fentes radiales (282) de la cage pour effectuer la rotation de la cage (281) et amener le palier (290) à se déplacer hors d'engagement avec le carter (295) et permettre la rotation continue de l'organe d'entraînement (252) dans le sens (B) pour effectuer la mise en action du mécanisme d'actionnement (90,230)

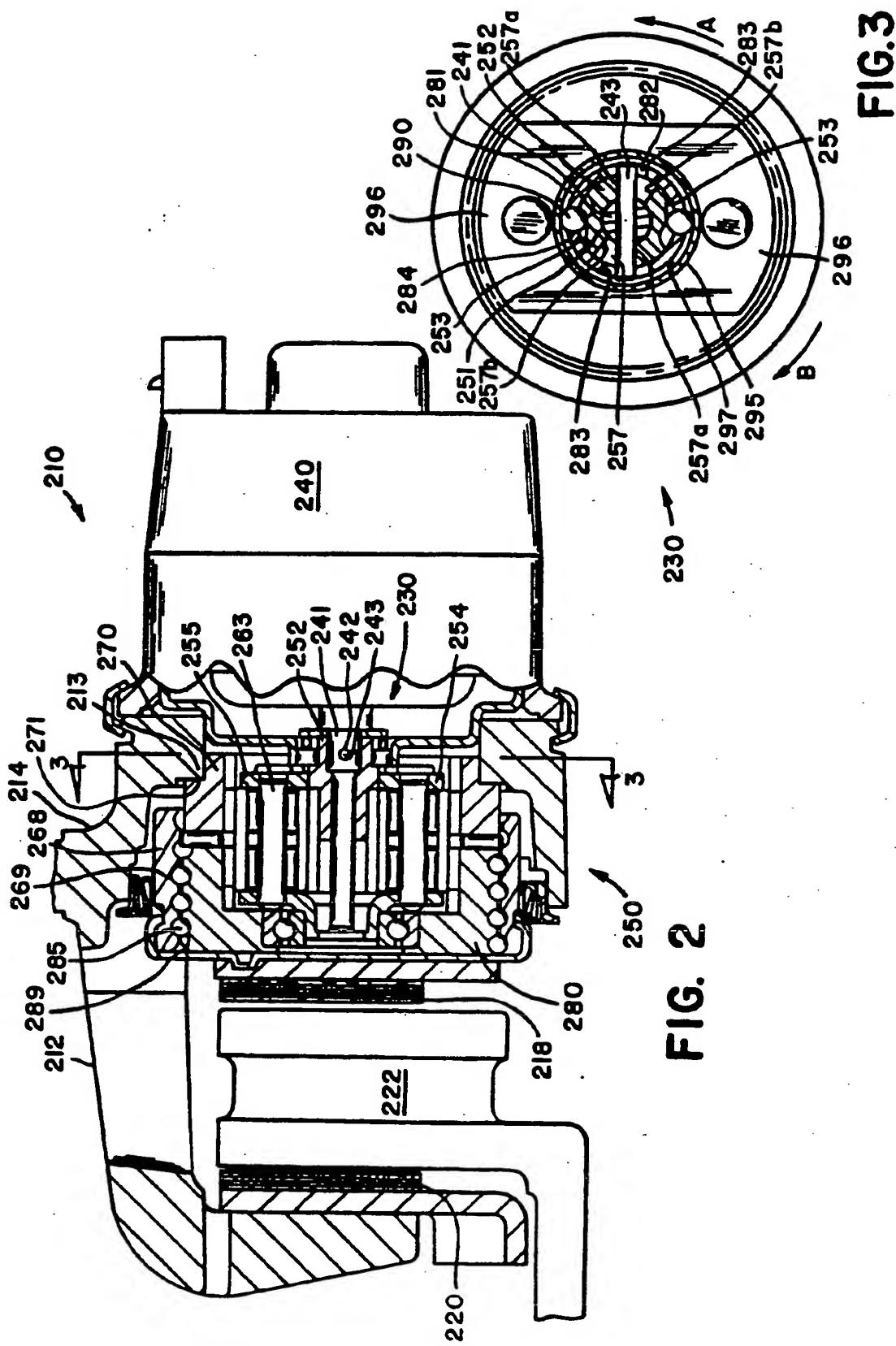
4. Frein à disque selon la revendication 3, dans lequel la cage (281)inclut une paire d'ouvertures (284) opposées dans lesquelles est disposé le palier (290) au contact des surfaces de cames (253) de l'organe d'entraînement (252).

5. Frein à disque selon la revendication 4, dans lequel les fentes radiales(282) de la cage sont disposées en non alignement circonférentiel avec les fentes d'entraînement (257).

6. Frein à disque selon la revendication 5, dans lequel les surfaces de cames (253) sont disposées radialement vers l'intérieur d'une surface périphérique extérieure de l'organe d'entraînement (252).

7. Frein à disque selon la revendication 6, dans lequel le palier (290) comprend une pluralité de roulements à galets.

8. Frein à disque selon la revendication 7, dans lequel l'organe d'entraînement -(252) inclut un alésage intérieur qui reçoit en rotation l'arbre d'entraînement (241).



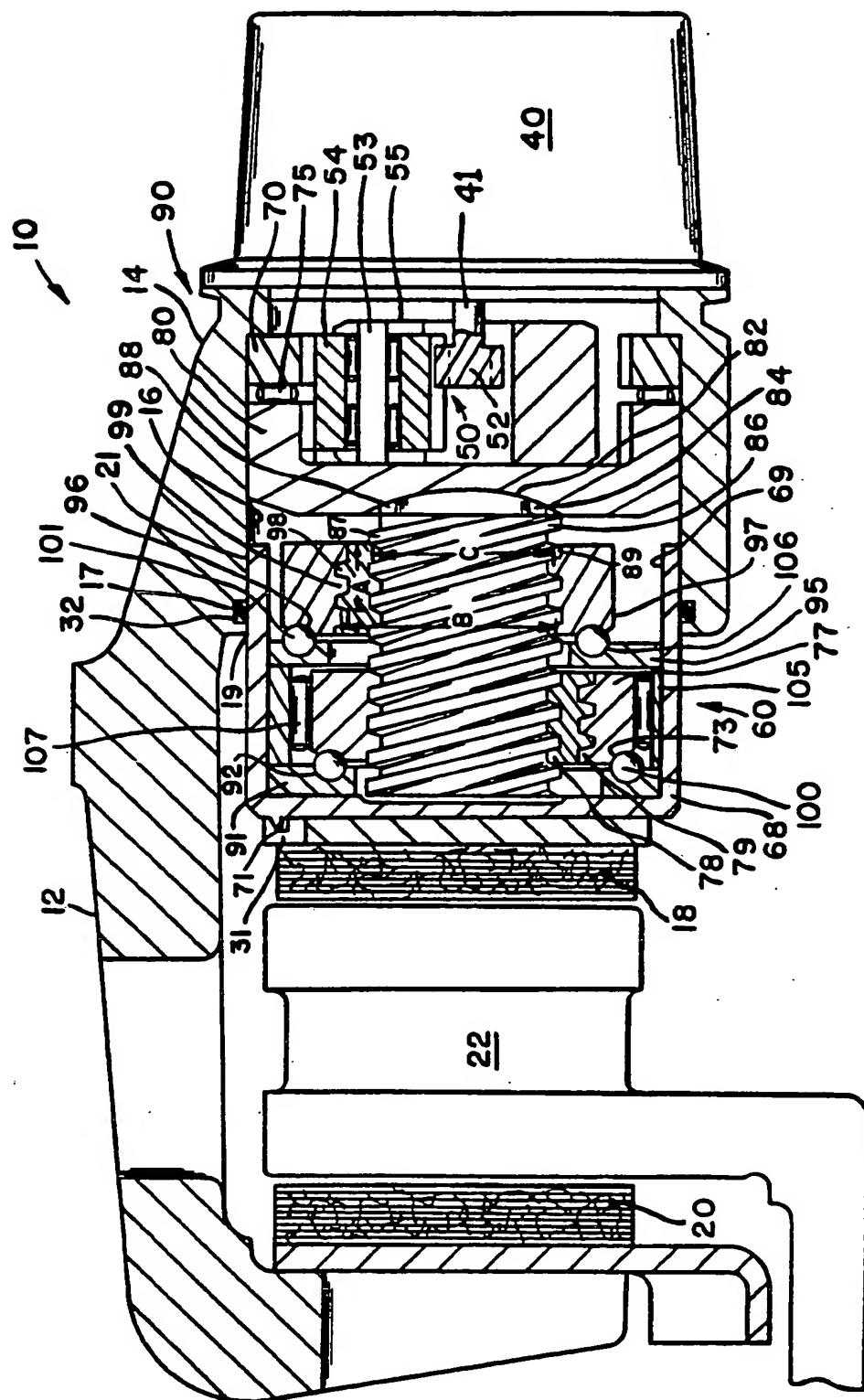


FIG. I